

V. Continuous

For B.C.:

$$\begin{aligned} M_z &= EI_z V'' \\ -Q_y &= (M_z)' = (EI_z V'')' \\ q &= -(Q_y)' = (EI_z V'')'' \end{aligned}$$

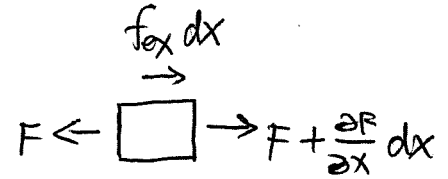
1. Longitudinal

$$\mu dx \frac{\partial^2 u}{\partial t^2} = \left[\left(F + \frac{\partial F}{\partial x} dx \right) + f_{ex} dx \right] - F$$

$$\Rightarrow \frac{\partial F}{\partial x} - \mu \frac{\partial^2 u}{\partial t^2} + f_{ex} = 0$$

$$\therefore \frac{F}{A} = E \frac{\partial u}{\partial x}$$

$$\Rightarrow \frac{\partial}{\partial x} \left(EA \frac{\partial u}{\partial x} \right) - \mu \frac{\partial^2 u}{\partial t^2} + f_{ex} = 0$$



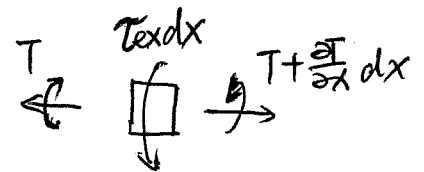
2. Torsional

$$I dx \frac{\partial^2 \theta}{\partial t^2} = \left[\left(T + \frac{\partial T}{\partial x} dx \right) + \tau_{ex} dx \right] - T$$

$$\Rightarrow \frac{\partial T}{\partial x} - I \frac{\partial^2 \theta}{\partial t^2} + \tau_{ex} = 0$$

$$\therefore T = GJ \frac{\partial \theta}{\partial x}$$

$$\Rightarrow \frac{\partial}{\partial x} \left(GJ \frac{\partial \theta}{\partial x} \right) - I \frac{\partial^2 \theta}{\partial t^2} + \tau_{ex} = 0$$



3. Flexural

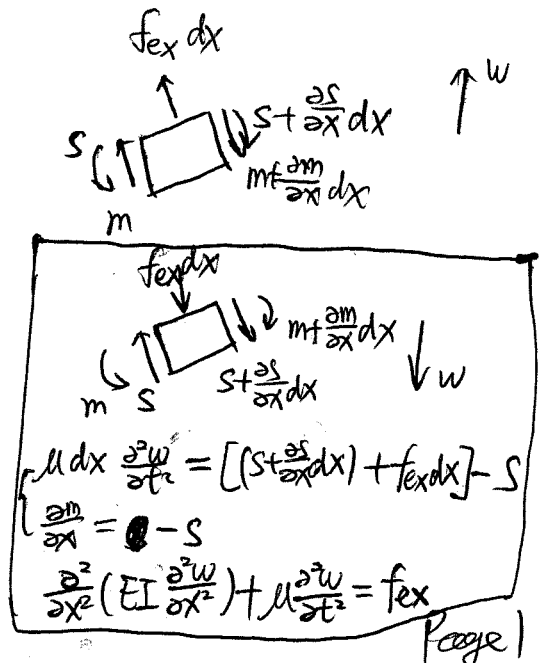
$$\mu dx \frac{\partial^2 w}{\partial t^2} = [S + f_{ex} dx] - \left[S + \frac{\partial S}{\partial x} dx \right]$$

$$\Rightarrow -\frac{\partial S}{\partial x} - \mu \frac{\partial^2 w}{\partial t^2} + f_{ex} = 0$$

$$\therefore \frac{\partial m}{\partial x} = -S$$

$$m = EI \frac{\partial^2 w}{\partial x^2}$$

$$\Rightarrow + \frac{\partial^2}{\partial x^2} \left(EI \frac{\partial^2 w}{\partial x^2} \right) - \mu \frac{\partial^2 w}{\partial t^2} + f_{ex} = 0$$



Transversal vibration solution:

(free vibration)

$$\frac{\partial^2}{\partial x^2} \left[EI(x) \frac{\partial^2 W(x,t)}{\partial x^2} \right] = -\mu(x) \frac{\partial^2 W(x,t)}{\partial t^2}$$

Using method of separation of variables,

$$W(x,t) = w(x) f(t)$$

$$\Rightarrow \frac{1}{\mu(x) w(x)} \frac{d^2}{dx^2} \left[EI(x) \frac{d^2 w(x)}{dx^2} \right] = -\frac{1}{f(t)} \frac{d^2 f(t)}{dt^2} = \omega^2$$

$$\therefore \begin{cases} \frac{d^2}{dx^2} \left[EI(x) \frac{d^2 w(x)}{dx^2} \right] - \omega^2 \mu(x) w(x) = 0 \\ \frac{d^2 f(t)}{dt^2} + \omega^2 f(t) = 0 \end{cases}$$

$$\text{Denote } \begin{cases} L = \frac{d^2}{dx^2} \left[EI(x) \frac{d^2}{dx^2} \right] \\ m = \mu(x) \\ \lambda = \omega^2 \end{cases}$$

$$\text{then } L[w] = \lambda m[w]$$

$w(x)$ is an eigenfunction of L associated with eigenvalue λ .

Orthogonality of Eigenfunctions =

If the problem is self-adjoint, i.e.

$$\begin{cases} \int_D w_s L[w_r] dD = \int_D w_r L[w_s] dD \\ \int_D w_s m[w_r] dD = \int_D w_r m[w_s] dD \end{cases}$$

where w_r and w_s are eigenfunctions associated with two distinct eigenvalues λ_r, λ_s .

$$\therefore \cancel{L[w_r]} L[w_r] = \lambda_r m[w_r]$$

$$L[w_s] = \lambda_s m[w_s]$$

$$\therefore \int_D w_s L[w_r] - w_r L[w_s] dD = \int_D \lambda_r w_s m[w_r] - \lambda_s w_r m[w_s] dD$$

$$\therefore 0 = (\lambda_r - \lambda_s) \int_D w_r m[w_s] dD$$

$$\therefore \int_D w_r m[w_s] dD = 0 \quad (\text{Orthogonality})$$

$\therefore w_r$ and w_s are orthogonal.

Expansion Theorem:
(orthonormal)

Eigenfunctions w_r form a complete set. ●

Any function w that satisfies the B.C. of the system and $L[w]$ is continuous, can be expanded by a convergent series of eigenfunctions:

$$w = \sum_{r=1}^{\infty} C_r w_r$$

where $C_r = \int_D w m[w_r] dD$, and

$$\int_D w_r m[w_r] dD = 1.$$

(Forced vibration)

$$-EI \frac{\partial^4 y(x,t)}{\partial x^4} + f(x,t) = m \frac{\partial^2 y(x,t)}{\partial t^2}$$

Suppose the free vibration problem has eigenfunctions $Y_r(x)$ ($r=1,2,\dots$),
associated with eigenvalues ω_r^2 .
orthonormal

$$\text{Let } y(x,t) = \sum_{r=1}^{\infty} Y_r(x) q_r(t)$$

$$\Rightarrow \sum_{r=1}^{\infty} m Y_r(x) \ddot{q}_r(t) + \sum_{r=1}^{\infty} EI \frac{d^4 Y_r(x)}{dx^4} q_r(t) = f(x,t)$$

$$\Rightarrow \ddot{q}_s(t) + \omega_s^2 q_s(t) = Q_s(t) \quad (s=1,2,\dots)$$

where generalized force,

$$Q_s(t) = \int_0^L f(x,t) Y_s(x) dx$$

q_s is the generalized coordinator. ?

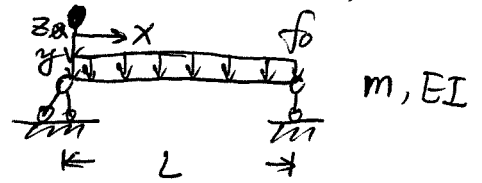
The generalized I.C. are.

$$\left\{ \begin{aligned} q_{s0} &= \int_0^L \cancel{m Y_s(x)} m Y_s(x) y_0(x) dx \\ \dot{q}_{s0} &= \int_0^L m Y_s(x) \dot{y}_0(x) dx \end{aligned} \right. \quad (s=1,2,\dots)$$

$$\therefore y(x,t) = \sum_{r=1}^{\infty} Y_r(x) \left[\frac{1}{\omega_r} \int_0^t Q_r(\tau) \sin \omega_r(t-\tau) d\tau + q_{r0} \cos \omega_r t + \frac{\dot{q}_{r0}}{\omega_r} \sin \omega_r t \right]$$

Continuous Vibration Problem I :

Uniform, simple supported beam, zero initial condition, Uniform load.



Differential equation for flexural vibration :

$$\frac{\partial^2}{\partial x^2} (EI \frac{\partial^2 W}{\partial x^2}) + m \frac{\partial^2 W}{\partial t^2} = F(t)$$

$$\Rightarrow EI \frac{\partial^4 W}{\partial x^4} + m \frac{\partial^2 W}{\partial t^2} = F(t) \quad (1)$$

I.C. & B.C. :

$$\begin{cases} W(x,0) = 0; \frac{\partial W}{\partial t}(x,0) = 0 \\ W(0,t) = 0; \frac{\partial^2 W}{\partial x^2}(0,t) = 0 \\ W(L,t) = 0; \frac{\partial^2 W}{\partial x^2}(L,t) = 0 \end{cases}$$

For free vibration

(2) B.C.

$$EI \frac{\partial^4 W}{\partial x^4} + m \frac{\partial^2 W}{\partial t^2} = 0 \quad \begin{cases} W(0,t) = 0 \\ \frac{\partial^2 W}{\partial x^2}(0,t) = 0 \\ W(L,t) = 0 \\ \frac{\partial^2 W}{\partial x^2}(L,t) = 0 \end{cases}$$

Suppose $W(x,t) = Y(x)q(t)$

normalize $Y_r(x)$:

$$\int_0^L m Y_r^2(x) dx = 1$$

$$\Rightarrow C_2 = \sqrt{\frac{2}{mL}}$$

$$\Rightarrow \frac{EI}{Y(x)} Y^{(4)}(x) + \frac{m}{q(t)} q^{(2)}(t) = 0$$

x, t independent

$$\Rightarrow \begin{cases} EI Y^{(4)}(x) - \omega^2 m Y(x) = 0 \\ \ddot{q}(t) + \omega^2 q(t) = 0 \end{cases} \quad (3) \text{ B.C. } \begin{cases} Y(0) = 0 \\ Y''(0) = 0 \\ Y(L) = 0 \\ Y''(L) = 0 \end{cases}$$

2. Eigen-value problem (4) have eigen gra

$$\omega_r^2 = \frac{EI}{m} \left(\frac{r\pi}{L} \right)^4 \quad (r=1,2,\dots)$$

$$Y_r(x) = \sqrt{\frac{2}{mL}} \sin \frac{r\pi x}{L}$$

2° For original problem, write

Let $L = EI \frac{d^4}{dx^4}$, $m = m$

$$W(x,t) = \sum_{r=1}^{\infty} Y_r(x) q_r(t)$$

$$\Rightarrow L[Y] = \omega^2 m[Y] \quad (4)$$

$$\Rightarrow L[W] + m \frac{\partial^2 W}{\partial t^2} = F(t)$$

Denote $\beta^4 = \frac{m\omega^2}{EI}$

$$\Rightarrow \sum_{r=1}^{\infty} L[Y_r] q_r(t) + \sum_{r=1}^{\infty} m[Y_r] \ddot{q}_r(t) = F(t)$$

$$\Rightarrow Y^{(4)}(x) - \beta^4 Y(x) = 0$$

Integrate $\int_0^L Y_r L dx = F(t)$

$$\Rightarrow Y(x) = C_1 \cos \beta x + C_2 \sin \beta x + C_3 \cosh \beta x + C_4 \sinh \beta x$$

$$\Rightarrow \omega_r^2 q_r(t) + \ddot{q}_r(t) = Q_r(t) \quad (5)$$

Using B.C.

$$Q_r(t) = \int_0^L Y_r(x) F(t) dx \quad \begin{cases} I.C. \\ q_{r0} = 0 \\ \dot{q}_{r0} = 0 \end{cases}$$

$$= \sqrt{\frac{2}{mL}} \int_0^L \sin \frac{r\pi x}{L} F(t) dx$$

$$\Rightarrow \begin{cases} Y_r(x) = \sin \beta_r x \\ \beta_r = \frac{r\pi}{L} \end{cases} \quad (r=1,2,\dots)$$

$$\begin{aligned}
 q_r(t) &= \int_0^t \frac{1}{\omega_r} \sin(\omega_r(t-\tau)) Q_r(\tau) d\tau \\
 &= \int_0^t \frac{1}{\omega_r} \sin(\omega_r(t-\tau)) \sqrt{\frac{2}{mL}} \frac{2}{\beta_r} f_0 u(\tau) d\tau \quad (r=1,3,5,\dots) \\
 &= \sqrt{\frac{2}{mL}} \frac{2f_0}{\beta_r \omega_r^2} (1 - \cos \omega_r t) u(t)
 \end{aligned}$$

$$\therefore W(x,t) = \sum_{r=1,3,5} \left[\frac{4f_0 L^4}{EI(r\pi)^5} \sin \frac{r\pi x}{L} \cdot \left(1 - \cos \sqrt{\frac{EI}{m}} \frac{r^2 \pi^2 t}{L^2} \right) \right] u(t)$$

□

Summary: 5 problems:

- 1° Original problem (IBC) (PDE)
- 2° Free vibration problem (B-C) (PDE)
- 3° Variable separate problem (B-C) (ODE)
- 4° Eigenvalue problem
- 5° Differential equation for generalized coordinate (IG) (ODE)