

第六章 不可压缩理想流体的平面势流问题.

- 总假设: 1° $\frac{D\rho}{Dt} = 0$; 2° $\mu = 0$; 3° $\vec{v} = \vec{v}(x,y) = u(x,y)\vec{i} + v(x,y)\vec{j}$; 4° $\vec{\omega} = 0$

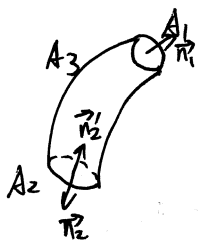
1. 理想流体的无旋运动.

1.1. 速度环量

$$\Gamma = \oint_L \vec{u} \cdot d\vec{r} = \oint_L u dx + v dy + w dz$$

$$\therefore \Gamma = \oint_A (\nabla \times \vec{u}) \cdot d\vec{A} = \int_A \vec{\omega} \cdot \vec{n} dA = I \quad (\text{涡通量}).$$

1.2. 涡管强度守恒定理.



在图示封闭曲面 $A_1 + A_2 + A_3 = A$ 上, 已知 A_3 为涡管面, A_1, A_2 任意,

则 $I_{A_1} = I_{A_2}$, 即涡量相等, 称涡管强度守恒.

$$I_A = \int_A \vec{\omega} \cdot d\vec{A} = \int_{\Omega} \nabla \cdot \vec{\omega} d\tau = \int_{\Omega} \nabla \cdot (\nabla \times \vec{v}) d\tau = 0$$

$$I_{A_3} = \int_{A_3} \vec{\omega} \cdot d\vec{A} = 0$$

$$\therefore \int_{A_1} \vec{\omega} \cdot \vec{n}_1 dA + \int_{A_2} \vec{\omega} \cdot \vec{n}_2 dA = 0$$

$$\therefore \int_{A_1} \vec{\omega} \cdot \vec{n}_1 dA = \int_{A_2} \vec{\omega} \cdot \vec{n}_2 dA$$

$$\therefore I_{A_1} = I_{A_2}$$

结论: 1° $A \downarrow \sim \omega \uparrow$

2° 涡管不能收缩为0; 涡管不能产生或终止; 在流体中只有涡环, 或始于终于边界 (这不同于涡旋不生不灭定理, 不要求无粘)

1.3. Kelvin's 速度环量定理.

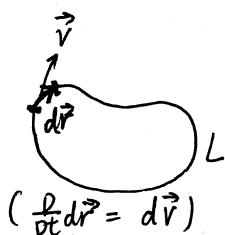
假设: 1° $\mu = 0$. 由欧拉方程

$$\frac{D\vec{u}}{Dt} = \frac{\partial \vec{u}}{\partial t} + (\vec{u} \cdot \nabla) \vec{u} = \vec{f}_b - \frac{\nabla P}{\rho}$$

$$2^\circ \nabla P = -\frac{\nabla P}{\rho}, 3^\circ -\nabla \Pi = \vec{f}_b$$

$$\text{得 } \frac{D\vec{u}}{Dt} = -\nabla(P + \Pi)$$

速度环量的随体导数:



$$\begin{aligned} \frac{D\Gamma}{Dt} &= \frac{D}{Dt} \oint_L \vec{u} \cdot d\vec{r} = \oint_L \frac{D}{Dt} (\vec{u} \cdot d\vec{r}) = \oint_L \frac{D\vec{u}}{Dt} \cdot d\vec{r} + \vec{u} \cdot \frac{D}{Dt} (d\vec{r}) \\ &= \oint_L \frac{D\vec{u}}{Dt} \cdot d\vec{r} + \oint_L d(\vec{u} \cdot \vec{r}) = 0 \end{aligned}$$

$$\therefore \frac{D\Gamma}{Dt} = -\oint \nabla(P + \Pi) \cdot d\vec{r} = -\oint d(P + \Pi) = 0$$

即为速度环量定理.

2° 由 $I = \Gamma$, 得 $\frac{DI}{Dt} = 0$, 即为 Helmholtz 涡管强度保持定理.

4.3. 物理法刚析法.

已知流体元: $\delta m, V$.沿流线: 迁移加速度 $v \frac{\partial v}{\partial s}$, 不定常加速度 $\frac{\partial v}{\partial t}$.
压差 Δp 法向速度梯度 $\frac{\partial v}{\partial n}$.相关物理量: $\rho, \nu, l, \mu, g, \Delta p, t_0 (\frac{1}{\omega})$

迁移惯性力: $F_i = \delta m \cdot v \frac{\partial v}{\partial s} \sim \rho l^2 v^2$

不定常惯性力: $F_{it} = \delta m \frac{\partial v}{\partial t} \sim \rho l^3 \frac{v}{t_0}$

重力: $F_g = \delta m g \sim \rho l^3 g$

粘性力: $F_v = \mu \frac{\partial v}{\partial n} \cdot \delta A \sim \mu \nu l$

压差力: $F_p = \Delta p \delta A \sim \Delta p l^2$

比较力的量级:

$$\frac{F_i}{F_v} = Re; \quad \frac{F_i}{F_g} = Fr^2; \quad \frac{F_p}{F_i} = Eu; \quad \frac{F_{it}}{F_i} = St.$$

$$Ma = \frac{v}{a} \left\{ \begin{array}{l} \leftarrow \text{流速} \\ \leftarrow \text{当地声速} \end{array} \right. \Rightarrow Ma^2 = \frac{v^2}{a^2} \quad (\text{由 } a^2 = \gamma RT \text{ 内能}) = \frac{\text{动能}}{\text{内能}}$$

$$\text{韦伯数 } We = \frac{\rho v^2 l}{\sigma} = \frac{\text{惯性力}}{\text{表面张力}}$$

5. 模型实验与相似性原理.

5.1. 模型实验

5.2. 相似性原理:

对相似的流动现象, 可以将模型实验的数据与结论定量的推广到原型流动.

原理: $\pi_1 = f(\pi_2, \pi_3, \dots, \pi_n)$ π 数与具体的尺寸参数无关, 也适用于模型实验.

$$\pi_{1m} = f(\pi_{2m}, \pi_{3m}, \dots, \pi_{nm})$$

若 $\pi_2 = \pi_{2m}, \dots, \pi_n = \pi_{nm}$

则 $\pi_1 = \pi_{1m}$.

3° 拉格朗日定理: (涡旋不生不灭定理)

假定 1°, 2°, 3° 成立时, 若流场某一部分无旋, 则在之前、之后该部分流体也无旋。

2. 速度势和流函数.

2-1. 速度势 若 $\vec{\omega} = 0$, 则存在 ϕ , 使得 $\vec{u} = \nabla\phi$. ($\nabla^2\phi = \nabla \cdot \vec{u}$)

对于等势线 $d\phi = 0$, 有 $\vec{u} \cdot d\vec{r} = 0$, 所以等势线上速度矢量

2-2. 流函数 (二维流场)

若流体不可压: $\nabla \cdot \vec{u} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$

可以找到 $\psi(x, y, t)$, 使

$$u = \frac{\partial \psi}{\partial y}, \quad v = -\frac{\partial \psi}{\partial x}$$

对极坐标下, $\nabla \cdot \vec{u} = \frac{1}{r} \frac{\partial}{\partial r}(ru_r) + \frac{1}{r} \frac{\partial}{\partial \theta}(u_\theta) = 0$

$$\therefore \text{流函数 } u_r = \frac{1}{r} \frac{\partial \psi}{\partial \theta}, \quad u_\theta = -\frac{\partial \psi}{\partial r}$$

对流函数的等值线: $d\psi = 0$,

$$\text{有 } \frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy = -v dx + u dy = 0$$

$$\therefore \frac{dx}{u} = \frac{dy}{v}, \text{ 即流函数等值线为流线.}$$

流量计算: $dQ = \vec{u} \cdot d\vec{l}$

$$= \vec{u} \cdot \vec{n} dl$$

$$= (u \frac{dy}{dl} - v \frac{dx}{dl}) dl$$

$$= u dy - v dx$$

$$= \frac{\partial \psi}{\partial y} dy + \frac{\partial \psi}{\partial x} dx$$

$$= d\psi$$

$$\therefore Q = \int_A^B dQ = \int_{\psi_A}^{\psi_B} d\psi = \psi_B - \psi_A$$

涡量-流函数方程.

$$\omega = \omega_z = \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} = -\frac{\partial^2 \psi}{\partial y^2} - \frac{\partial^2 \psi}{\partial x^2} = -\nabla^2 \psi$$

$\therefore \nabla^2 \psi = -\omega_z$ 为 2D 不可压缩流体的 ω - ψ 方程.

3. 平面势流与基本解.

假设: 1° $\mu = 0$ 2° $\nabla \cdot \vec{v} = 0$ 3° $\vec{u}(x, y) = u(x, y)\vec{i} + v(x, y)\vec{j}$
4° $\frac{\partial}{\partial t} \equiv 0$ 5° $\vec{\omega} = 0$

由 2°, 5° 得 $\nabla^2 \phi = 0, \nabla^2 \psi = 0$

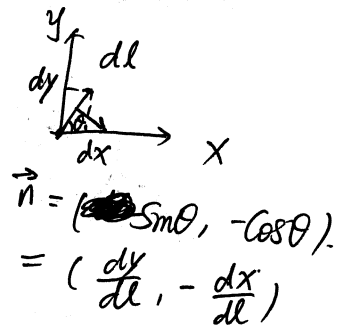
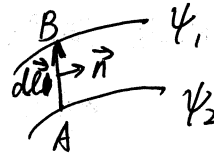
说明平面势流中的速度势和流函数都是调和函数.

速度势的唯一性是显然的任意一个无旋的矢量场, 按路径积分的方法, 在一个常数的区别下, 唯一的确定了它的势函数.

流函数的唯一性:

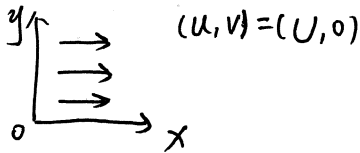
当流体可压时, 流函数显然不存在。即流体不可压是流函数存在的必要条件。

当流体不可压时, 相当于已知 $\nabla \cdot \vec{u} = 0$, 一个无旋的矢量场唯一地确定了它的势函数在不考虑一个常数的区别下。所以不可压流体的流场存在, 且在同样的意义下, 唯一存在一个流函数。



求解平面势流的基本解, 是为了构造复杂的, 或实际的流动.

3.1. 均匀来流问题



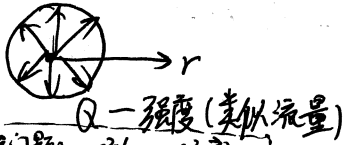
$$\text{由 } \frac{\partial \psi}{\partial y} = u, \quad \frac{\partial \psi}{\partial x} = -v$$

$$\therefore \psi = Uy + C_1$$

$$\text{由 } \frac{\partial \phi}{\partial x} = u, \quad \frac{\partial \phi}{\partial y} = v$$

$$\therefore \phi = Ux + C_2$$

3.2. 点源(汇) ($Q = \text{Constant}$)



定解问题: $\nabla^2 \phi = Q \delta^2(r)$
 $\nabla^2 \psi = 0$

$$\text{由 } u_r = \frac{\partial \phi}{\partial r} = \frac{Q}{2\pi r}, \quad u_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = 0$$

$$\therefore \phi = \frac{Q}{2\pi} \ln r + C_1$$

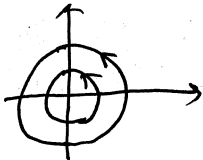
$$\text{由 } u_r = \frac{1}{r} \frac{\partial \psi}{\partial \theta} = \frac{Q}{2\pi r}, \quad u_\theta = -\frac{\partial \psi}{\partial r} = 0$$

$$\therefore \psi = \frac{Q}{2\pi} \theta + C_2$$

3.3. 点涡 ($\Gamma = \text{Constant}$)

涡线的强度 $\Gamma = 2\pi r v_\theta = \text{Constant}$.

$$\therefore v_\theta = \frac{\Gamma}{2\pi r}$$



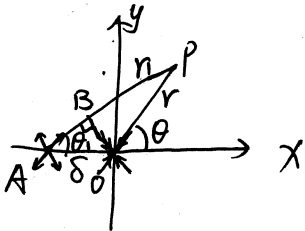
$$\text{由 } u_r = \frac{\partial \phi}{\partial r} = 0, \quad u_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = \frac{\Gamma}{2\pi r}$$

$$\therefore \phi = \frac{\Gamma}{2\pi} \theta$$

$$\text{由 } u_r = \frac{1}{r} \frac{\partial \psi}{\partial \theta} = 0, \quad u_\theta = -\frac{\partial \psi}{\partial r} = \frac{\Gamma}{2\pi r}$$

$$\therefore \psi = -\frac{\Gamma}{2\pi} \ln r$$

3.4. 偶极子 (汇(点)与源(A点)强度均为 Q)



叠加可得流函数:

$$\psi_p(r, \theta) = \frac{Q}{2\pi} \theta_1 - \frac{Q}{2\pi} \theta_2 \quad (Q > 0)$$

$$\therefore \overline{OB} = \delta \sin \theta_1$$

$$\therefore \theta - \theta_1 = \frac{\delta \sin \theta_1}{r} \quad (\text{极限情况})$$

$$\therefore \psi(r, \theta) = \lim_{\delta \rightarrow 0} -\frac{Q}{2\pi} \frac{\delta \sin \theta_1}{r} = -\frac{Q}{2\pi} \frac{\delta \sin \theta}{r}$$

定义偶极矩 $M = Q\delta$

$$\therefore \psi(r, \theta) = -\frac{M}{2\pi} \frac{\sin \theta}{r}$$

$$\text{同理 } \phi(r, \theta) = \frac{M}{2\pi} \frac{1}{\delta} \ln r_1 - \frac{M}{2\pi} \frac{1}{\delta} \ln r_2$$

$$= \frac{M}{2\pi} \frac{1}{\delta} \ln \left(1 + \frac{\delta \cos \theta_1}{r} \right)$$

$$= \frac{M}{2\pi} \frac{1}{\delta} \frac{\delta \cos \theta}{r}$$

$$= \frac{M}{2\pi} \frac{\cos \theta}{r}$$

定解问题:

$$\begin{cases} \nabla^2 \phi = Q(-\delta^2 \vec{r}) + \delta^2(\vec{r} + \delta \vec{e}_1) \\ (\text{即 } Q\delta = M, \delta \rightarrow 0) \\ \nabla^2 \psi = 0 \end{cases}$$

可叠加性(系统线性性):

因为 3.2 点源(汇)的定解问题是线性的, 由叠加原理, 偶极子的 ϕ, ψ 可由 3.2 的解叠加而成。

定解性:

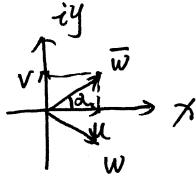
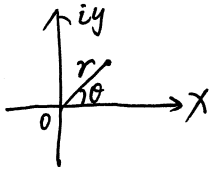
$$\nabla \phi = \nabla \cdot \vec{u}, \quad \nabla \psi = -\vec{u}_z$$

给定一矢量场的散度和旋度, 由 Helmholtz 定理, 除了一个常数的差别, 矢量场是

(流场) 唯一确定的. 进而 ϕ, ψ 也在同样意义下被确定。

4. 平面势流的复势

4.1. 复势和复速度



$$\begin{cases} u = \frac{\partial \phi}{\partial x} = \frac{\partial \psi}{\partial y} \\ v = \frac{\partial \phi}{\partial y} = -\frac{\partial \psi}{\partial x} \end{cases}$$

$\therefore \phi, \psi$ 满足 Cauchy-Riemann 条件.

\therefore 存在 $F(x, y) = \phi + i\psi$, 称为平面势流的复势.

又定义 $W(z) = \frac{dF}{dz} = \frac{\partial F}{\partial x} = \frac{\partial \phi}{\partial x} + i \frac{\partial \psi}{\partial x} = u - iv$, 称为复速度.

$$\begin{aligned} \bar{W}(z) &= u + iv, \text{ 称为共轭复速度.} \\ &= |\bar{w}| e^{i\alpha} \end{aligned}$$

$$\therefore u = \operatorname{Re} \left\{ \frac{dF}{dz} \right\}, \quad v = -\operatorname{Im} \left\{ \frac{dF}{dz} \right\}$$

$$\text{积为 } \oint_L W(z) dz = \oint_L \frac{dF}{dz} dz = \oint_L dF = \oint_L \phi + i \oint_L \psi = \Gamma + iQ.$$

$$\begin{cases} \Gamma = \oint_L \vec{u} \cdot d\vec{l} = \oint_L \nabla \phi \cdot d\vec{r} = \oint_L d\phi \\ Q = \oint_L dQ = \oint_L d\psi \end{cases}$$

ϕ, ψ 的环路积不一定为零, 因为 ϕ, ψ 可以是不对的函数, 具体参见教材.

4.2. 平面势流的复势解法.

1° 均匀直线流

$$F(z) = Cz$$

$$W(z) = C = u - iv$$

若 $C = U_0 e^{-i\alpha}$, 则 $u = U_0 \cos \alpha, v = U_0 \sin \alpha$.

2° 点源 (汇)

$$F(z) = A \ln z = A \ln r + iA\theta$$

$$v_r = \frac{\partial \phi}{\partial r} = \frac{A}{r}$$

$$v_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = 0$$

$$\text{环量 } \Gamma = \operatorname{Re} \left\{ \oint_L W(z) dz \right\} = \operatorname{Re} \left\{ \oint_L \frac{A}{z} dz \right\} = \operatorname{Re} \{ 2\pi i \cdot A \} = 0$$

$$\text{流量 } Q = \operatorname{Im} \left\{ \oint_L W(z) dz \right\} = 2\pi A$$

$$\therefore A = \frac{Q}{2\pi}, \quad F_1(z) = \frac{Q}{2\pi} \ln z$$

3° 点涡

$$F(z) = iA \ln \bar{z} = iA \ln(re^{-i\theta}) = -A\theta + iA \ln r$$

$$v_r = \frac{\partial \phi}{\partial r} = 0$$

$$v_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = -\frac{A}{r}$$

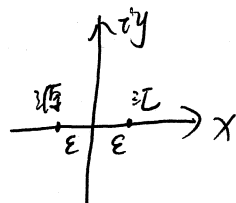
$$\text{流量 } Q = \operatorname{Im} \left\{ \oint_L W(z) dz \right\} = \operatorname{Im} \left\{ \oint_L \frac{iA}{z} dz \right\} = \operatorname{Im} \{ -2\pi A \} = 0$$

$$\text{环量 } \Gamma = \operatorname{Re} \left\{ \oint_L W(z) dz \right\} = -2\pi A$$

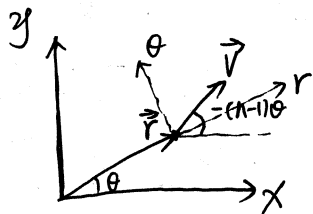
$$\therefore A = -\frac{\Gamma}{2\pi}, \quad F(z) = i \left(-\frac{\Gamma}{2\pi} \right) \ln \bar{z} = \frac{\Gamma}{2\pi i} \ln \bar{z}$$

4° 偶极子

$$\begin{aligned}
 F(z) &= \frac{Q}{2\pi} \ln(z+\epsilon) - \frac{Q}{2\pi} \ln(z-\epsilon) \\
 &= \frac{Q}{2\pi} \ln\left(\frac{z+\epsilon}{z-\epsilon}\right) \\
 &= \frac{Q}{2\pi} \ln\left(\frac{1+\frac{\epsilon}{z}}{1-\frac{\epsilon}{z}}\right) \\
 &= \frac{Q}{2\pi} \ln\left[\left(1+\frac{\epsilon}{z}\right)\left(1+\frac{\epsilon}{z}+o\left(\frac{\epsilon}{z}\right)\right)\right] \\
 &= \frac{Q}{2\pi} \ln\left[1+2\frac{\epsilon}{z}+o\left(\frac{\epsilon}{z}\right)\right] \\
 &= \frac{Q}{2\pi} \left(\frac{2\epsilon}{z}\right) \quad (M \equiv \lim_{\epsilon \rightarrow 0} \lim_{Q \rightarrow \infty} 2QE) \\
 &= \frac{M}{2\pi} \frac{1}{z}
 \end{aligned}$$



5° 角域流



$$F(z) = Az^n \quad (n \geq \frac{1}{2}, n \neq 1, A \text{ 为实数})$$

$$W(z) = Anz^{n-1} = Anr^{n-1}e^{i\theta(n-1)} = u - iv$$

$$\bar{W}(z) = A \cdot n r^{n-1} e^{-i(n-1)\theta}$$

\therefore ~~到x轴~~ x 轴到 \vec{r} 的角为 θ .
到 \vec{v} 的

$$\therefore r \text{ 轴到 } \vec{v} \text{ 的角 } [(n-1)\theta] - \theta = -n\theta$$

$$\therefore \begin{cases} u_r = Anr^{n-1} \cos n\theta \\ v_\theta = -Anr^{n-1} \sin n\theta \end{cases}$$

$$F(z) = Ar^n e^{in\theta} = Ar^n \cos n\theta + i Ar^n \sin n\theta$$

令 $\psi = 0$, (称为零流线)

$$\therefore Ar^n \sin n\theta = 0$$

$$\therefore n\theta = k\pi, \text{ 即 } \theta = k\frac{\pi}{n} = k\Delta\theta.$$

① 若 $n=3$, 则 $\Delta\theta = \frac{\pi}{3}$

$$u_r = 3Ar^2(-1)^k$$



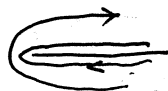
② $n=\frac{3}{2}$, 则 $\Delta\theta = \frac{2\pi}{3}$

$$u_r = \frac{3}{2}Ar^{\frac{1}{2}}(-1)^k$$



③ $n=\frac{1}{2}$, 则 $\Delta\theta = 2\pi$

$$u_r = \frac{1}{2}Ar^{-\frac{1}{2}}(-1)^k.$$



④ $n < \frac{1}{2}$, 则 $\Delta\theta > 2\pi$, 无解.

⑤ $n=2$ 时, $\Delta\theta = \frac{\pi}{2}$

$$u_r = 2Ar(-1)^k$$



6° 无环量圆柱绕流 (直线流 + 偶极子)

$$F(z) = U_\infty z + \frac{M}{2\pi} \frac{1}{z}$$

$$= (U_\infty r \cos\theta + \frac{M}{2\pi r} \cos\theta) + i(U_\infty r \sin\theta - \frac{M}{2\pi r} \sin\theta)$$

零流线 $\psi = 0$.

$$\therefore U_\infty r \sin\theta - \frac{M}{2\pi r} \sin\theta = 0$$

$$\therefore \begin{cases} \theta = 0, \pi \\ r = \sqrt{\frac{M}{2\pi U_\infty}} = a \end{cases}$$

$$\therefore \frac{M}{2\pi} = U_\infty a^2$$

$$\therefore F(z) = U_\infty z + U_\infty \frac{a^2}{z} \quad (|z| \geq a)$$

$$W(z) = U_\infty (1 - \frac{a^2}{z^2}) = U_\infty (1 - \frac{a^2}{r^2} e^{-i2\theta}) = U_\infty (e^{i\theta} - \frac{a^2}{r^2} e^{-i\theta}) e^{-i\theta}$$

$$= [U_\infty (1 - \frac{a^2}{r^2}) \cos\theta + i U_\infty (1 + \frac{a^2}{r^2}) \sin\theta] e^{-i\theta}$$

$$= (u_r - i u_\theta) e^{-i\theta}$$

~~驻点~~ $(\pm a, 0)$ 为两个驻点.

由伯努利积分 $p_\infty + \frac{1}{2} \rho U_\infty^2 = p_s + \frac{1}{2} \rho U_s^2$ (平面势流假设下, 伯努利积分成立.)

$$\text{压强系数 } C_p = \frac{p_s - p_\infty}{\frac{1}{2} \rho U_\infty^2} = 1 - \frac{U_s^2}{U_\infty^2} = 1 - \frac{U_\theta^2}{U_\infty^2} = 1 - 4 \sin^2\theta. \quad (p_s \text{ 为物面压强})$$

驻点压力

7° 有环量圆柱绕流

$$F(z) = U_\infty z + \frac{M}{2\pi} \frac{1}{z} + \frac{\Gamma}{2\pi i} \ln z \quad (\Gamma \text{ 表示顺时针环量})$$

$$= U_\infty (z + \frac{a^2}{z}) + \frac{\Gamma}{2\pi i} \ln z$$

$$= U_\infty (r + \frac{a^2}{r}) \cos\theta + i U_\infty (r - \frac{a^2}{r}) \sin\theta - \frac{\Gamma}{2\pi i} (\ln r + i\theta)$$

$$= (U_\infty (r + \frac{a^2}{r}) \cos\theta - \frac{\Gamma}{2\pi} \theta) + i [U_\infty (r - \frac{a^2}{r}) \sin\theta + \frac{\Gamma}{2\pi} \ln r]$$

$$u_r = \frac{\partial \phi}{\partial r} = U_\infty (1 - \frac{a^2}{r^2}) \cos\theta$$

$$u_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = -U_\infty (1 + \frac{a^2}{r^2}) \sin\theta - \frac{\Gamma}{2\pi r}$$

当 $r = a$ 时, $u_r = 0$, $u_\theta = -2U_\infty \sin\theta - \frac{\Gamma}{2\pi a}$

$$\therefore \text{驻点为 } (a, \theta_c), \sin\theta_c = -\frac{\Gamma}{4\pi a U_\infty}$$

① $\Gamma = 0$, $\sin\theta_c = 0$, 无环量问题

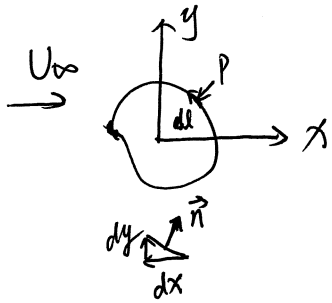
② $\Gamma < 4\pi U_\infty a$, $-1 < \sin\theta_c < 0$, 驻点下移

③ $\Gamma = 4\pi U_\infty a$, $\theta_c = -\frac{\pi}{2}$

④ $\Gamma > 4\pi U_\infty a$, 驻点分离.

5. 柱体定常流中的力与力矩

5.1 布拉修斯 (Blasius) 定理: 合力公式



$$d\vec{F} = -p\vec{n} dl$$

$$\Rightarrow \begin{cases} dF_x = -p \left(\frac{dy}{dl}\right) dl = -p dy \\ dF_y = -p \left(-\frac{dx}{dl}\right) dl = p dx \end{cases}$$

$$\therefore d(F_x - iF_y) = -ip(dx - idy)$$

由伯努利积分 $p_\infty + \frac{1}{2}\rho U_\infty^2 = p + \frac{1}{2}\rho U_s^2$

0 合力是 $F_x - iF_y$ (合力)
(U_s : ~~物面~~ 物面流速)

$$\therefore p = p_\infty + \frac{1}{2}\rho U_\infty^2 - \frac{1}{2}\rho U_s^2$$

$$= p_0 - \frac{1}{2}\rho U_s^2$$

其中 $p_0 = p_\infty + \frac{1}{2}\rho U_\infty^2$ 为驻点压强

$$\therefore U_s^2 = \cancel{|\vec{w}|^2} = \vec{w}\bar{w} = \frac{dF}{dz} \left(\frac{d\bar{F}}{d\bar{z}}\right) = \frac{dF}{dz} \frac{d\bar{F}}{d\bar{z}}$$

$$\therefore p = p_0 - \frac{1}{2}\rho \frac{dF}{dz} \frac{d\bar{F}}{d\bar{z}}$$

$$\therefore d(F_x - iF_y) = \frac{1}{2}ip \frac{dF}{dz} \left(\frac{d\bar{F}}{d\bar{z}}\right) d\bar{z} - ip_0 d\bar{z}$$

$$= \frac{1}{2}ip \left(\frac{dF}{dz}\right)^2 d\bar{z} - ip_0 d\bar{z}$$

(\because 柱面上 $d\psi = 0$)

$$\therefore \text{总力 } F_x - iF_y = \oint_L d(F_x - iF_y)$$

$$c \frac{dF}{dz} = (d\phi + id\psi) = d\phi = d\phi = dF$$

$$= \oint_L \frac{1}{2}ip \left(\frac{dF}{dz}\right)^2 d\bar{z} - ip_0 d\bar{z}$$

(~~之解法~~ 环量不为0) ($\oint_L d\bar{z} = \oint_L \bar{z} dz = 0$)

$$= \frac{1}{2}ip \oint_L \left(\frac{dF}{dz}\right)^2 d\bar{z}$$

$$= \frac{1}{2}ip \oint_L w^2(z) dz$$

5.2. 儒可夫斯基定理

复势 $F(z)$ 或复速度 $W(z)$ 在物面以外解析, 可以在^{在原点}将其展开成罗朗级数: 0

$$W(z) = \frac{dF(z)}{dz} = \dots + \frac{a_{-1}}{z} + a_0 + a_1 z + \dots$$

$$\text{其中 } a_{-1} = \frac{1}{2\pi i} \oint_L \frac{dF}{dz} dz = \frac{1}{2\pi i} (\Gamma + iQ)$$

$$\left(\oint_L \frac{dF}{dz} dz = \oint_L (u - iv)(dx + idy) \right)$$

$$\text{又 } W(z)|_\infty = \left. \frac{dF}{dz} \right|_\infty = U_\infty e^{-i\alpha}$$

$$= \oint_L (u dx + v dy) + i \oint_L (u dy - v dx)$$

$$\therefore a_1 = a_2 = \dots = 0$$

$$= \oint_L \vec{v} \cdot d\vec{r} + i \oint_L \vec{v} \times d\vec{r}$$

$$\therefore W(z) = \frac{dF}{dz} = U_\infty e^{-i\alpha} + \frac{Q - i\Gamma}{2\pi} \frac{1}{z} + \frac{a_2}{z^2} + \dots$$

$$= \Gamma + iQ$$

$$\therefore F_x - iF_y = \frac{1}{2}ip \oint_L \left(U_\infty e^{-i\alpha} + \frac{Q - i\Gamma}{2\pi} \frac{1}{z} + \dots \right)^2 dz$$

$$= \frac{1}{2}ip \cdot 2\pi i \cdot 2U_\infty \frac{-i\Gamma}{2\pi} e^{-i\alpha} \quad (\text{其中物面上 } Q=0)$$

$$= \rho U_\infty \Gamma i e^{-i\alpha}$$

$$= \rho U_\infty \Gamma i (\cos\alpha - i\sin\alpha)$$

$$\therefore F_x = \rho U_\infty \Gamma \sin\alpha$$

$$\Rightarrow \vec{F} = \rho U_\infty \times \vec{n}$$

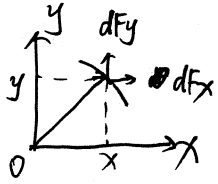
(Γ 以逆时针为正)

讨论: ① 理想不可压流体绕任一剖面的柱体作无旋体绕流中, 物体所受流体作用合力大小

与 ρ, U_∞, Γ 成正比

② 若 $\vec{v} = 0$, 则 $\vec{F} = \vec{0}$, 称为达朗贝尔佯谬, 是因为没有考虑粘性。

5.3. 布拉修斯第二定理: 合力矩公式.



$$\begin{aligned} dM &= dF_y x - dF_x y \\ &= \operatorname{Re} [i (dF_x - i dF_y) z] \\ &= \operatorname{Re} [i (-p dy - i p dx) z] \\ &= \operatorname{Re} [p (dx - i dy) z] \\ &= \operatorname{Re} [p z d\bar{z}] \end{aligned}$$

$$\begin{aligned} \therefore M &= \operatorname{Re} \left[\oint_L \left(p_0 - \frac{1}{2} \rho \frac{dF}{dz} \frac{d\bar{F}}{d\bar{z}} \right) z d\bar{z} \right] \\ &= \operatorname{Re} \left[\oint_L p_0 z d\bar{z} \right] - \operatorname{Re} \left[\oint_L \frac{1}{2} \rho \left(\frac{dF}{dz} \right)^2 z d\bar{z} \right] \\ &= -\frac{\rho}{2} \operatorname{Re} \left[\oint_L \left(\frac{dF}{dz} \right)^2 z d\bar{z} \right] \\ &= -\frac{\rho}{2} \operatorname{Re} \left[\oint_L \left(U_\infty e^{-i\alpha} + \frac{\Gamma}{2\pi i} \frac{1}{z} + \frac{a_{-2}}{z^2} + \dots \right)^2 z d\bar{z} \right] \\ &= -\frac{\rho}{2} \operatorname{Re} \left[\oint_L \left(2\pi i (2U_\infty e^{-i\alpha} a_{-2} - \frac{\Gamma^2}{4\pi^2}) \right) \right] \\ &= -2\pi \rho U_\infty \operatorname{Re} [i e^{-i\alpha} a_{-2}] \end{aligned}$$

$$\begin{aligned} \operatorname{Re} \left[\oint_L p_0 z d\bar{z} \right] &= p_0 \oint_L x dx + y dy \\ &= \frac{1}{2} \oint_L dr^2 \\ &= 0 \end{aligned}$$